

Pump @ model

The most general Pump model is given as a function of the mass flowrate on the intake p_{in} and discharge pressure p_{out} :

$$(1) \quad \dot{m} = M(p_{out}, p_{in})$$

It's often presented in terms of intake volumetric flowrate:

$$(2) \quad q = q_{in} = \frac{\dot{m}}{\rho(p_{in})} = \frac{M(p_{out}, p_{in})}{\rho(p_{in})}$$

where

| | |
|-----------|---|
| $\rho(p)$ | fluid density as a function of fluid pressure p |
|-----------|---|

The electrical power consumption $W = \frac{dE}{dt}$ is given by:

$$(3) \quad W = \eta(q) \cdot q \cdot (p_{out} - p_{in})$$

where

| | |
|--------|-----------------|
| η | pump efficiency |
|--------|-----------------|

In most practical cases the pump model (2) depends on the difference between intake and discharge pressure $p_{out} - p_{in}$ and called Pump Characteristic Curve (see Fig. 1):

$$(4) \quad q = q(p_{out} - p_{in})$$

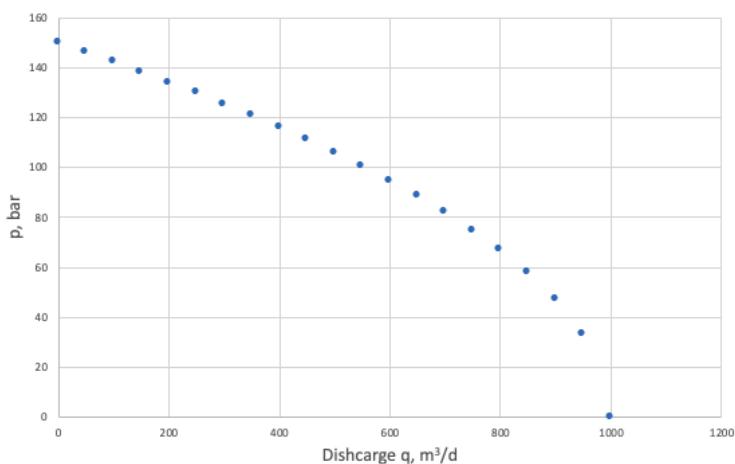


Fig. 1. Example of Pump Characteristic Curve.

A popular pump proxy model is given by the quadratic equation with 3 inputs ($\{q_{rfmax}, \delta p_{max}, k_f\}$):

$$(5) \quad q = \frac{q_{max}}{2 \cdot k_f} \cdot \left[-1 + k_f + \sqrt{(1 + k_f)^2 - 4 \cdot k_f \cdot (p_{out} - p_{in})/\delta p_{max}} \right]$$

$$(6) \quad p_{out} = p_{in} + \delta p_{max} \cdot \left[1 + (k_f - 1) \cdot \frac{q}{q_{max}} - k_f \cdot \left(\frac{q}{q_{max}} \right)^2 \right]$$

$$(7) \quad \eta(q) = 4 \eta_{max} \cdot q/q_{max} \cdot (1 - q/q_{max})$$

where

| | |
|-------------------|--|
| δp_{\max} | maximum pressure gain that pump can exert over the input pressure p_{in} |
| q_{\max} | maximum flowrate that pump can produce |
| $k_f \in [0, 1]$ | total hydraulic pump friction (dimensionless) |
| η | pump efficiency |
| η_{\max} | maximum pump efficiency |

Real pumps have non-constant $k_f = k_f(q)$ friction coefficient which often modelled as a 3rd order polynomial and the overall real-pump model taking 6-inputs.

Many pumps can be normally adjusted by the variation of the working frequency which affects the maximum [pump flow rate](#) and maximum pressure gain as:

$$(8) \quad q_{\max} = q_{\max}^* \cdot \frac{f}{f^*}$$

$$(9) \quad \delta p_{\max} = \delta p_{\max}^* \cdot \left(\frac{f}{f^*} \right)^2$$

where

| | | | | | |
|--------------|--|---------------------|--|-------|----------------------------|
| q_{\max} | maximum intake flowrate at the working frequency f | δp_{\max} | maximum pressure gain at the working frequency f | f | adjusted working frequency |
| q_{\max}^* | maximum intake flowrate at the nominal frequency f^* | δp_{\max}^* | maximum pressure gain at the nominal frequency f^* | f^* | nominal frequency |

See also

[Natural Science / Engineering / Device / Pump](#)

[Physics / Fluid Dynamics / Pipe Flow Dynamics / Pipe Flow Simulation \(PFS\)](#)

References
